

#### Transient Characteristics of GDI Injectors using Statistical Extinction Tomography



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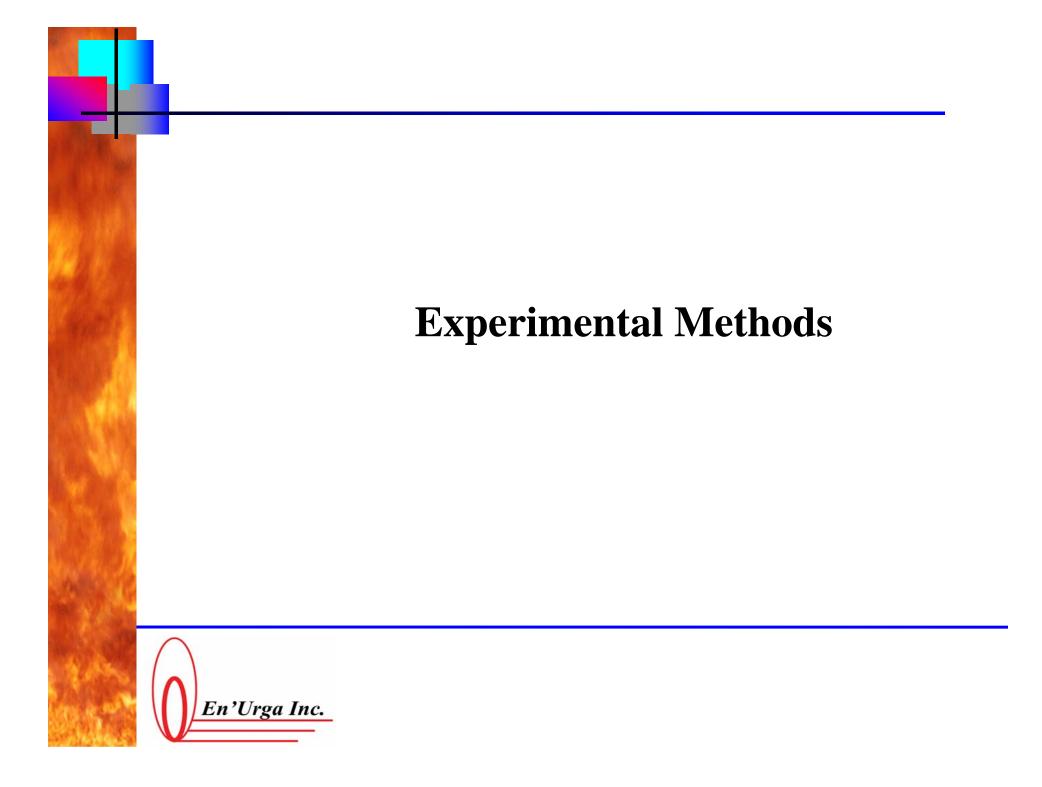


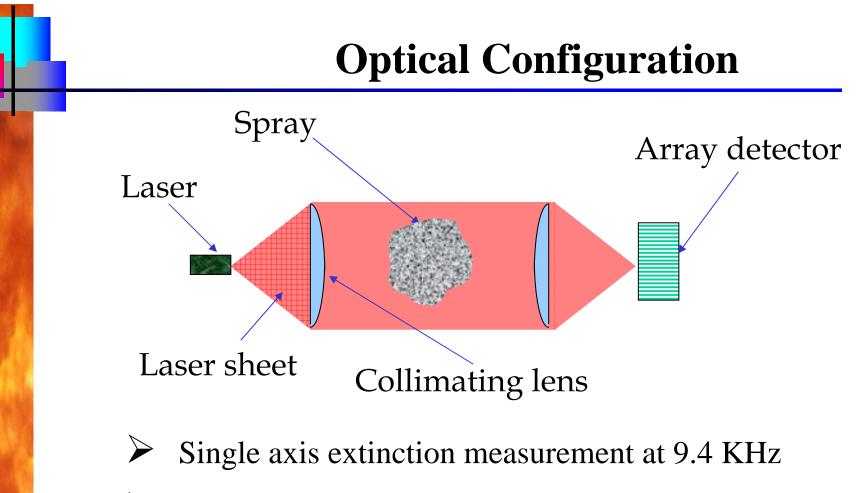
# Outline Background > Experimental methods $\succ$ Results En'Urga Inc.

- Spray characteristics affect engine performance (Speigel and Spicher, 1992, Davy et al., 2000, Zhoa et al., 2002)
- Multi-hole configuration better than pressure swirl atomizers (Fujieda et al., 1995; Arcoumanis et al., 1998)
- Spray characteristics vary with ambient pressure and fuel temperature (Schmitz et al., 2002; Parish et al., 2010)

Objective of this study is to investigate the use of extinction tomography to characterize dense multiple plume sprays







- Planar extinction through 512 parallel paths
  - Injector rotated 12 times to obtain multiple view data



## **Experimental Arrangement**



#### **Setscan AP400 patternator**

- 250 mm nominal ID
- 100 mm silica windows
- ~ 1m high vessel to reduce spray bounce
- Injector rotated by 15 deg for a total of 12 views
- Inert nitrogen atmosphere
- **Fuel heated upto 90** °C
- 5 injections per angle

#### **Principle of Operation**

- Path integrated extinction of laser sheets
- > 12 view angles, 15 degrees apart
- ➤ 5 injection events ensemble averaged at each view angle
- ➢ 512 parallel path measurements at each view angle
- Local extinction coefficients in the plane obtained by deconvolution (MLE – Vardi and Lee, 1993)
- ➢ For dielectric liquids, local extinction coefficient is proportional to the drop surface areas per unit volume



#### **Importance of Surface Area**

#### **Density**

Control volume

Drop size varied from 1 to 100 micron Velocity varied from 1 to 10 m/s Number of drops varied from 1E5 to 1E7 Calculate evaporation rate

For static drop

 $\dot{m} = \frac{4\pi k_g r_s}{C_{pg}} \ln(B+1)$  $\dot{m}_{dynamic} = \frac{N_u}{2} \dot{m}_{static}$  $B = \frac{C_{pg}(T_\alpha - T_b)}{h_{fg}}$ 

For dynamic drop

$$B = \frac{pg < \alpha - b^{2}}{h_{fg}}$$

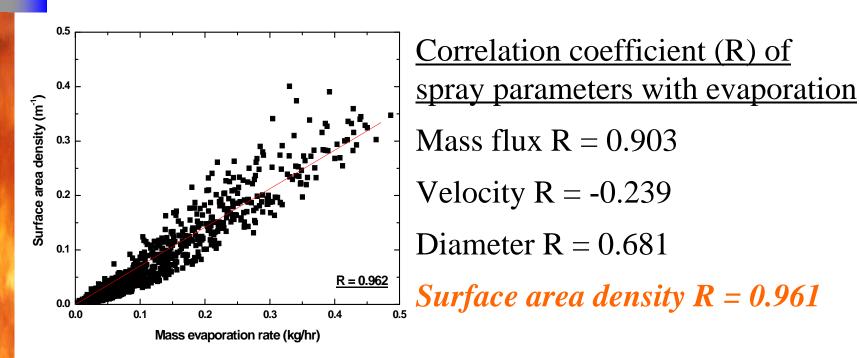
$$C_{pg} = C_{pF}(\overline{T}) \text{ and } k_{g} = 0.4k_{F}(\overline{T}) + 0.6k_{\infty}(\overline{T}); \quad \overline{T} = (T_{b} + T_{\infty})/2$$

$$N_{u} = 2 + \frac{0.555 \operatorname{Re}^{1/2} \operatorname{Pr}^{1/3}}{\left[1 + 1.232/(\operatorname{Re} \operatorname{Pr})^{4/3}\right]^{1/2}}$$

\*An Introduction to Combustion: Concepts and Applications," Stephen R. Turns, Mc. Graw Hill, 2000



# **Importance of Surface Area Density**



- Total amount of fuel or liquid evaporated is proportional to heat release rate in combustion
- Important for combustion and spray drying

#### **Test Matrix**

Test Conditions	Ambient Pressure (kPa)	Fuel Temperature (°C)	Injection Pressure (MPa)		
1	101	20	15		
2	40	90	5		
3	60	60	5		
4	60	60	10		
5	101	20	10		
6	40	90	2		

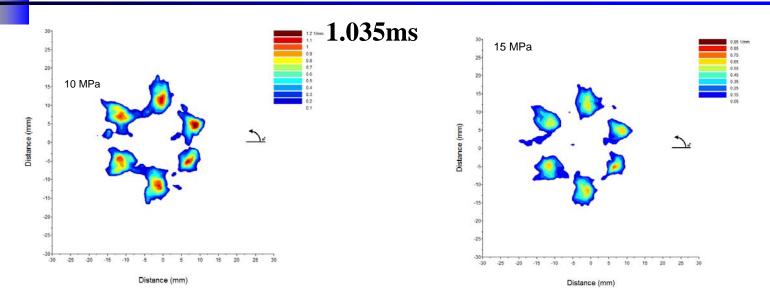
Injection at higher temperatures and lower than atmospheric pressures to simulate engine conditions. All tests carried out with calibrated gasoline-E10



#### Results



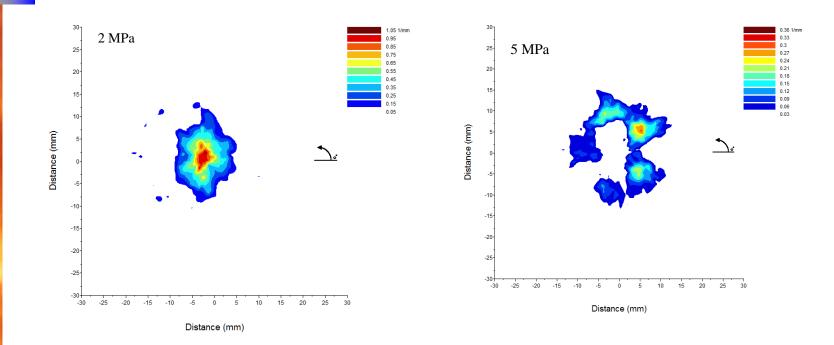
## Six Plume Injector (20<sup>o</sup> C)



Plume ID	Center <i>R</i> ( <i>mm</i> )	Center $\theta$ ( <i>deg</i> )	Center X ( <i>mm</i> )	Center Y ( <i>mm</i> )	Contour Area $(mm^2)$	Plume Angle ( <i>deg</i> )	Peak Area ( <i>mm</i> <sup>-1</sup> )	Total Area ( <i>mm</i> <sup>2</sup> )	% in Plume
1	12.1	19.8	8.0	5.1	33.2	7.4	0.69	11.3	13.2
2	11.8	77.5	-0.9	12.5	45.5	8.7	0.69	16.3	19.0
3	10.6	142.1	-11.8	7.5	43.6	8.5	0.69	14.7	17.1
4	10.3	218.0	-11.6	-5.3	47.4	8.9	0.75	15.4	17.9
5	12.6	278.0	-1.7	-11.5	44.8	8.6	0.71	16.4	19.0
6	11.8	330.8	6.8	-4.7	27.5	6.8	0.83	9.4	11.0
Total (pl	Total (plume separation at 15.9%)								97.2



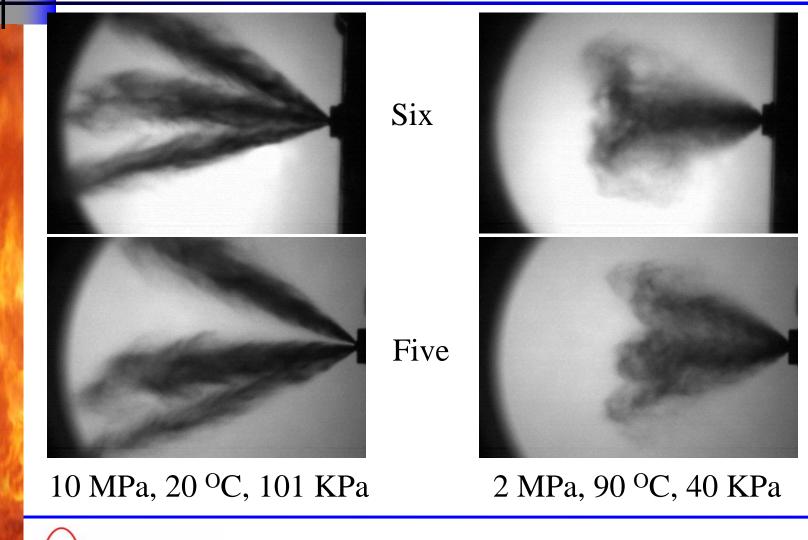
#### Six Plume Injector (90<sup>o</sup> C)



#### Individual plumes not visible at low ambient pressures and high fuel temperatures

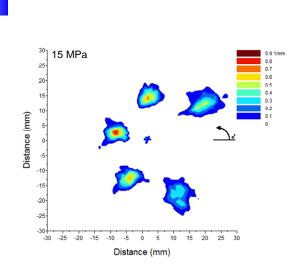


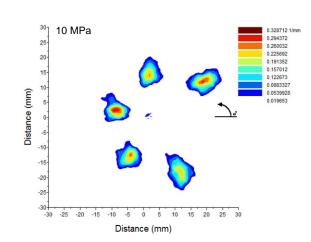
#### **High Speed Video**



## Five Plume Injector (20<sup>0</sup> C)

1.035ms

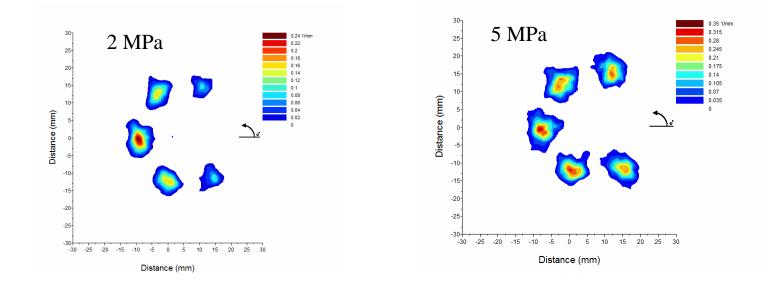




Plume ID	Center R (mm)	Center θ (deg)	Center X (mm)	Center Y (mm)	Contour area (mm2)	Plume angle (deg)	Peak area (mm-1)	Total area (mm2)	% in plume
1	22.7	35.6	18.5	13.2	108.8	26.5	0.31	10.7	22.2
2	14.9	86.1	1.0	14.9	89.3	24.1	0.26	8.3	17.1
3	9.3	165.8	-9.0	2.3	103.4	25.8	0.32	9.3	19.3
4	13.6	253.4	-3.9	-13.0	99.8	25.4	0.28	8.6	17.8
5	21.3	305.6	12.4	-17.3	131.3	29.0	0.26	10.9	22.6
Total (plume separation at 5.3%)							47.8	99.1	

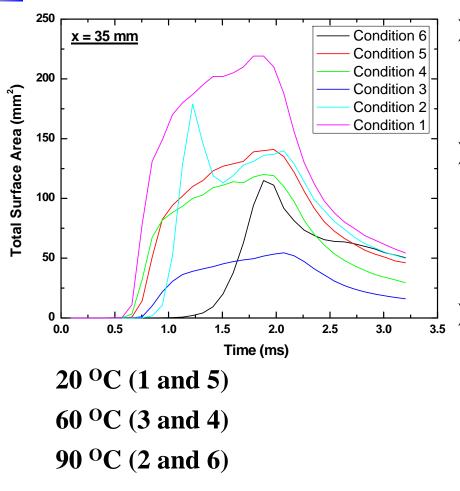






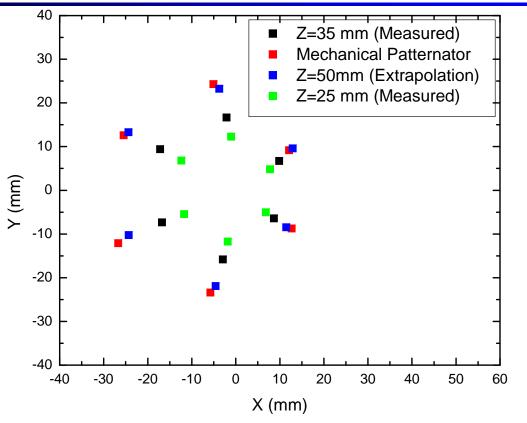
Plumes still distinct at higher temperatures and lower ambient pressure





- Profiles similar for same temperature and ambient pressures
- Lower injection
   pressures shows delay in
   peak values and lower
   peak values
- Injection temperature changes profile significantly

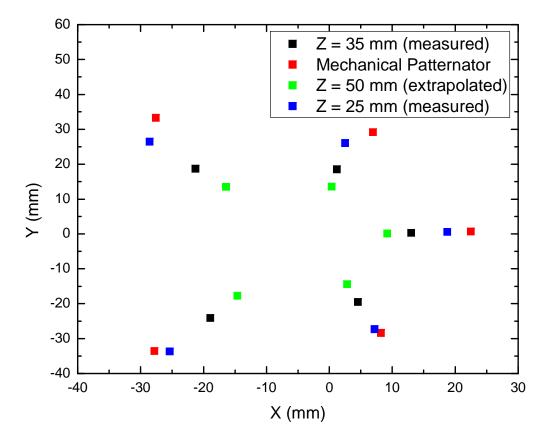
## **Comparison with Mechanical Patternator**



Centriods from mechanical patternator is slightly further away



## **Comparison with Mechanical Patternator**



Centriods from mechanical patternator is slightly further away



- The peak surface area density increasing with increasing injection pressure.
- There is a merging of the plumes at higher temperatures and lower ambient pressures.
- The centroid locations obtained from the mechanical and optical patternator were in good agreement for both the injectors tested in this study
- > The centroid locations do not vary much with pressure
- Extinction tomography is useful to characterizing the fuel spray from GDI injectors

